

# Wind Shear Estimation by Frequency-Shaped Optimal Estimator

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This paper presents a formulation of the frequency-shaped optimal estimator and investigates the role of the measurement noise shaping function in this estimator. This shaping function creates transmission zeros and affects the singular values of the estimator transfer matrix, which can be used to improve control system robustness. The resulting estimator design technique has been applied to the estimation of wind shear onboard an airplane, and its performance has been simulated and compared with a reference Kalman filter.

## Nomenclature

$A$	= system dynamics matrix
$A_v, B_v, C_v, D_v$	= realization matrices of measurement noise shaping function
$A_w, B_w, C_w, D_w$	= realization matrices of process noise shaping function
$a_{hm}$	= mean horizontal wind shear
$\hat{a}_{hm}$	= estimate of $a_{hm}$
$B_c$	= control distribution matrix
$C$	= measurement distribution matrix
$c_t$	= thrust coefficient
$F(s)$	= shaping filter transfer matrix
$f_t$	= wind turbulence characteristic frequency
$G_c$	= feedback control gain matrix
$K, K_w, K_v$	= gains of frequency-shaped estimator
$Q$	= spectral density matrix of process noise
$Q^{1/2}(s)$	= process noise shaping function
$R$	= spectral density matrix of measurement noise
$R^{1/2}(s)$	= measurement noise shaping function
$r, r'$	= estimator residuals
$r_m$	= ground speed measurement
$s$	= Laplace transform frequency variable
$T_F(s)$	= estimator transfer matrix
$t$	= time
$U_0$	= equilibrium airspeed
$u_a$	= perturbed airspeed measurement
$u_c$	= control vector
$V_{hm}$	= mean horizontal wind speed
$V_{ht}$	= turbulence in horizontal wind
$v, v'$	= measurement noise vector, Gaussian white noise vector
$w, w'$	= process noise vector, Gaussian white noise vector
$\chi(\cdot)$	= stability derivatives for aircraft longitudinal axis
$x, x_w, x_v, x_z$	= state vectors of system and shaping filters
$\hat{x}, \hat{x}_w, \hat{x}_v, \hat{x}'$	= estimates of state vectors
$y'$	= output vector
$Z(\cdot)$	= stability derivatives for aircraft vertical axis
$z$	= measurement vector
$\delta_e$	= elevator angle, deg

$\eta_{ah}, \eta_{at}$	= process noise components of wind dynamics
$\theta$	= perturbed pitch angle
$v_{ua}, v_f$	= measurement noise for airspeed and ground speed
$\Phi(s)$	= system dynamics transfer matrix
$\omega$	= frequency

## Introduction

AUTOLAND control systems designed to maintain airspeed in the presence of wind shear tend to have large bandwidths that admit significant amounts of noise from wind turbulence through the airspeed measurement. This noise can produce unacceptable levels of control activity, particularly in thrust control. Previous research<sup>1</sup> has shown that onboard estimation of wind shear is possible by means of a steady-state Kalman filter; however, sufficiently responsive estimates still contain significant noise due to the first-order roll off characteristics of the Kalman filter.

Gupta<sup>2</sup> has proposed the use of frequency-shaped cost functionals in optimal estimator design to improve the frequency response characteristics of the estimate. These functionals result in additional dynamic states in the estimator, but these states need not be fed to the regulator in the complete control loop, so various estimator formulations could be used with the same regulator design.

Application of Gupta's approach to wind shear estimation requires careful selection of the frequency shaping functions since a robust estimator is necessary due to the inability to accurately define a dynamic model for all wind shear conditions. Safonov<sup>3</sup> and others have discussed the importance of the singular values of the system return difference matrix in determining system robustness. A reformulation of Gupta's noise-shaping transfer functions, described in this paper, shows that the poles of the measurement noise shaping function become transmission zeros of the estimator transfer matrix, which permits some control of the system singular values. Thus, robust estimation with good high-frequency attenuation can be achieved.

Subsequent sections describe the formulation of the frequency-shaped estimator, its application to wind shear modeling for a phugoid approximation of a typical transport airplane, and its simulated performance in a speed-hold system.

## Frequency-Shaped Optimal Estimator

Consider the linear system:

$$\begin{aligned}\dot{x} &= Ax + B_c u_c + w \\ z &= Cx + v\end{aligned}\quad (1)$$

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As Gupta shows (Ref. 2 and references therein), the assumption of frequency-shaped cost functionals in the performance index can be implemented by treating  $w$  and  $v$  as autocorrelated noise sources generated by shaping filters of the form

$$\begin{aligned} w(j\omega) &= Q^{1/2}(j\omega)w'(j\omega) \\ v(j\omega) &= R^{1/2}(j\omega)v'(j\omega) \end{aligned} \quad (2)$$

where  $Q$  and  $R$  are functions of the complex frequency  $j\omega$  such that

$$\begin{aligned} Q(j\omega) &= Q^{1/2}(j\omega)[Q^{1/2}(j\omega)]^* \\ R(j\omega) &= R^{1/2}(j\omega)[R^{1/2}(j\omega)]^* \end{aligned}$$

Here  $Q^{1/2}$  and  $R^{1/2}$  are rational functions of  $j\omega$  (i.e. transfer matrices) and  $[\cdot]^*$  represents complex conjugation.

Gupta shows that for systems with states of finite amplitude,  $Q^{1/2}$  must represent proper transfer functions (i.e., having at least as many poles as zeros), so that it can be realized by a dynamic system of the form

$$\begin{aligned} \dot{x}_w &= A_w x_w + B_w w' \\ w &= C_w x_w + D_w w' \end{aligned} \quad (3)$$

However,  $R^{1/2}(j\omega)$  is not similarly restricted; indeed, strictly proper transfer functions in  $R^{1/2}$  would cause  $R$  to approach zero at high frequency, implying perfect measurements in that frequency range. Thus, for systems with realistic sensor models, the inverse filter  $R^{-1/2}(j\omega)$  should be proper, and the measurement equations from Eq. (1) can be expressed, using Eq. (2), as

$$R^{-1/2}(j\omega)z(j\omega) = R^{-1/2}(j\omega)Cx(j\omega) + v'(j\omega) \quad (4)$$

Define a pseudomeasurement as

$$z'(j\omega) \equiv R^{-1/2}(j\omega)z(j\omega)$$

and a noise-free output as

$$y'(j\omega) \equiv R^{1/2}(j\omega)Cx(j\omega)$$

Then a dynamic realization of  $R^{-1/2}$  can be formed as

$$\begin{aligned} \dot{x}_v &= A_v x_v + B_v Cx \\ y' &= C_v x_v + D_v Cx \end{aligned} \quad (5)$$

with pseudomeasurement written as

$$z' = C_v x_v + D_v Cx + v' \quad (6)$$

Combining Eqs. (1), (3), (5), and (6) defines a dynamic system driven by independent white noise sources for which a constant-gain estimator (i.e., a steady-state Kalman filter) can be designed. This estimator can be represented by the equations

$$\begin{aligned} \dot{\hat{x}} &= A\hat{x} + C_w \hat{x}_w + K[z' - C_v \hat{x}_v - D_v C\hat{x}] + B_c u_c \\ \dot{\hat{x}}_w &= A_w \hat{x}_w + K_w [z' - C_v \hat{x}_v - D_v C\hat{x}] \\ \dot{\hat{x}}_v &= A_v \hat{x}_v + B_v C\hat{x} + K_v [z' - C_v \hat{x}_v - D_v C\hat{x}] \end{aligned} \quad (7)$$

where  $z'$  is obtained from  $z$ , using the realization of  $R^{-1/2}(j\omega)$  in Eq. (5), by the equations

$$\begin{aligned} \dot{x}_z &= A_z x_z + B_z z \\ z' &= C_z x_z + D_z z \end{aligned} \quad (8)$$

Thus, the designer's problem is to select  $Q^{1/2}(j\omega)$  and  $R^{-1/2}(j\omega)$ , construct their realizations in Eqs. (3) and (5), and then design the estimator in Eq. (7) to meet performance requirements. The resulting estimator algorithm can be implemented by combining Eqs. (7) and (8). Letting  $\hat{x}' = x_z - \hat{x}_v$ , the implementation becomes

$$\begin{aligned} \dot{\hat{x}} &= (A - KD_v C)\hat{x} + C_w \hat{x}_w + KC_v \hat{x}' + KD_v z + B_c u_c \\ \dot{\hat{x}}_w &= A_w \hat{x}_w + K_w C_v \hat{x}' - K_w D_v C\hat{x} + K_w D_v z \\ \dot{\hat{x}}' &= (A_v - K_v C_v)\hat{x}' - (B_v - K_v D_v)C\hat{x} + (B_v - K_v D_v)z \end{aligned} \quad (9)$$

where measurements  $z$  are the input, and only  $\hat{x}$  is fed to the regulator using, for example, the control law  $u_c = G_c \hat{x}$ . While techniques such as linear-quadratic synthesis can be used to obtain gains  $K$ ,  $K_w$ ,  $K_v$ , and  $G_c$ , the question of how to select the shaping filters remains. Since shaping filters add dynamic states to the estimator, the designer must trade performance and complexity in selecting the filters. Criteria for this selection can be developed based upon the effect of these filters on estimator frequency response; and, thus, on control loop robustness.

### Transmission Zeros, Singular Values, and Robustness

For multivariable systems, frequency response characteristics are not as easily defined as for single-input/single-output (SISO) systems. However, theoretical efforts have been made to relate system robustness (i.e., margins of stability) to the magnitudes of the singular values<sup>4</sup> of return difference matrices.<sup>3,5</sup> These vary with frequency, and their values are influenced by the presence of transmission zeros in the loop. (Transmission zeros are the values of  $s$  at which an input signal produces no output.<sup>6</sup>) It can be shown, as is done in detail in Ref. 7, that measurement noise shaping can be used to insert transmission zeros in a multivariable loop at specified frequencies, altering the robustness of the loop.

This effect can be seen by taking the Laplace transform of Eq. (9) and formulating the block diagram of Fig. 1, where

$$r = z - C\hat{x} \quad \text{and} \quad r' = D_v r + C_v \hat{x}$$

Here, the transfer matrices for the system and each of the shaping filters, as obtained from Eqs. (1), (3), and (5), are represented by

$$\Phi(s) \equiv (sI - A)^{-1}$$

and, similarly, for  $\Phi_w(s)$  and  $\Phi_v(s)$ . Note that the shaping filters form cascade elements for the standard estimator of Eq. (1), i.e. if  $Q^{1/2}$  and  $R^{1/2}$  are identity matrices and  $K_w = K_v = 0$ , then  $r' = r$  and the steady-state Kalman filter results.

The feedback of  $K_v$  alters the poles of  $\Phi_v(s)$ , but the numerator polynomials of the transfer matrix from  $r$  to  $r'$  are preserved and thus appear in  $T_F(s)$ , the estimator transfer from  $z$  to  $\hat{x}$ . For the SISO case, this polynomial represents the zeros of  $R^{-1/2}$  (i.e. the poles of  $R^{1/2}$ ); and for the multivariable case, these polynomials can be shown to satisfy the definition of transmission zeros when  $R^{-1/2}$  is diagonal.<sup>7</sup> To see this, Fig. 1 can be rearranged as shown in Fig. 2 where the measurement prefilter of Eq. (8) has been separated from the shaping filters in Eq. (7), which are represented by  $F(s)$ . Then, setting  $G_c$  to zero for convenience,

$$\hat{x}(s) = \Phi(s)F(s)R^{-1/2}(s)[z(s) - C\hat{x}(s)]$$

so that

$$T_F(s) = [I + \Phi(s)F(s)R^{-1/2}(s)C]^{-1}\Phi(s)F(s)R^{-1/2}(s)$$

Thus,  $R^{-1/2}$  is a factor of  $T_F$  and, if it is diagonal with each element representing a SISO shaping function for a particular measurement, the zeros of  $r_i^{-1/2}(s)$  will appear in the



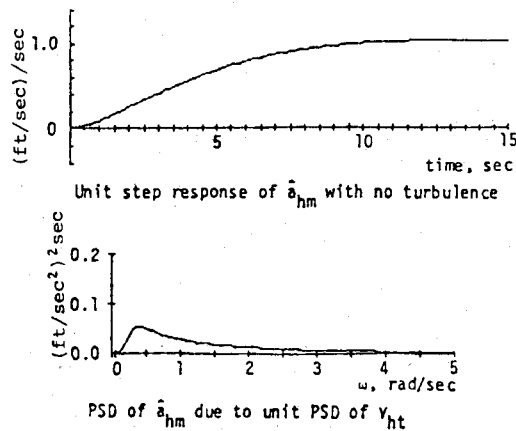


Fig. 3 Performance of the reference Kalman filter.

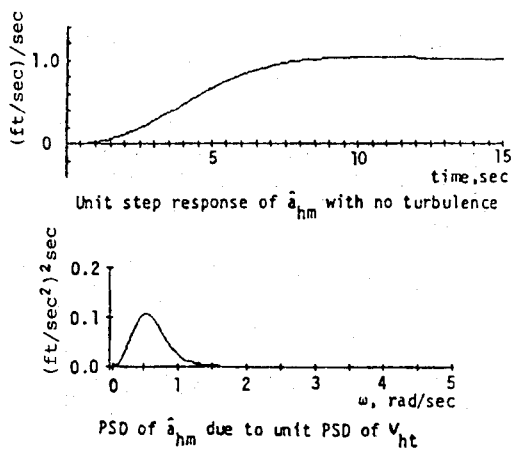


Fig. 4 Performance of the frequency-shaped estimator.

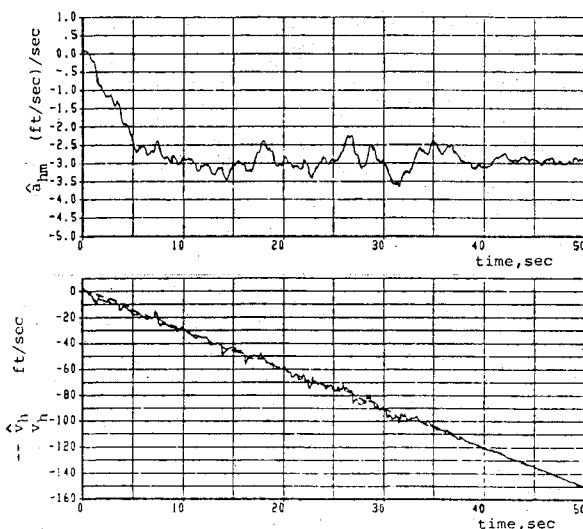


Fig. 5 Simulated performance of the reference Kalman filter.

Table 2 Estimator gain matrices

Ref. Kalman	Frequency-shaped			
	K		$K_V$	
-0.048    0.266	-0.272	0.333	0.723	-0.082
0.058    -0.063	0.241	-0.084	0.812	-0.122
-1.119    0.313	-2.250	0.508	-0.045	0.163
-0.309    0.024	-0.573	0.063		
-0.665    0.0005	-0.241	0.013		

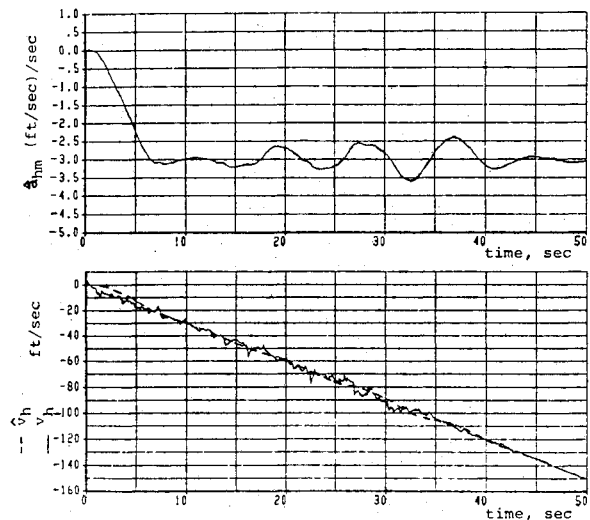


Fig. 6 Simulated performance of the frequency-shaped estimator.

Table 3 Control gain matrix  $G_c$ 

-0.34	-3.96	0.34	0	0
-1.36	-4.06	1.36	3.0	0

frequency of the numerator determines the intensity of the noise assumed in the high frequency range.

Thus, since the phugoid frequency is roughly the operating range of the speed-hold system, the inverse of the shaping function for the first measurement noise has been selected to be

$$r_1^{-1/2}(s) = (s^2 + 2.8s + 4.0)/16(s^2 + 0.7s + 0.25)$$

which has corner frequencies at  $\omega = 0.5$  and  $2$  rad/s and, therefore, assumes the noise to have a magnitude at higher frequency 16 times greater than at lower frequency.

Shaping the first measurement only with the above function has proved to provide good noise attenuation at high frequency. Further study<sup>7</sup> suggested that control loop robustness could be further improved by minor shaping applied to the second measurement, using a 1st-order function

$$r_2^{-1/2}(s) = (s + 1.5)/3(s + 0.5)$$

which assumes the noise to have three times greater magnitude at higher frequency than at lower frequency. The performance of the estimator using these noise shaping functions is shown in Fig. 4, which shows significant improvement in PSD of  $\hat{a}_{hm}$  over the reference Kalman filter while maintaining the equivalent time response.

Figures 5 and 6 show the simulated performance of the reference Kalman filter and the frequency-shaped optimal estimator described above. Each estimator (see Table 2) has been inserted in a simulation of the autopilot system using the phugoid model of Eqs. (10-12) and a linear-quadratic regulator (see Table 3). For this simulation,

$$V_{hm} = -3t \text{ ft/s}$$

has been assumed, with the von Kármán turbulence of non-Gaussian distribution,<sup>10,11</sup> which represents a severe wind condition for landing approach.<sup>1</sup> As is clear from these simulation results, the frequency-shaped estimator exhibits a much smoother wind shear estimate while keeping the time response characteristics equivalent to those of the reference Kalman filter. A more extensive example for a full-order aircraft model is given in Ref. 7.

### Conclusions

The results of this study indicate that frequency shaping of the measurement noise for an onboard estimation of wind shear can be effective in reducing the high frequency noise in the estimate while retaining good time response. These results are achieved by assigning transmission zeros of the estimator transfer function to an appropriate frequency band, which also influences control system robustness. The process of determining desired transmission zeros, based on desired noise attenuation, aids the selection of shaping functions to be used in the estimator. The relation of these functions to control loop robustness through the singular values of the return difference matrices is currently being studied.

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